AMENDMENTS TO THE SPECIFICATION

Please replace the paragraph beginning at Col. 3, Line 18, with the following paragraph:

The upper surface of main bearing housing 24 is provided with an annular flat thrust bearing surface 54 on which is disposed an orbiting scroll member 56. Scroll member 56 comprises an end plate 58 having the usual spiral vane or wrap 60 on the upper surface thereof and an annular flat thrust surface 62 on the lower surface thereof. Projecting downwardly from the lower surface is a cylindrical hub 64 having a journal bearing 66 therein and in which is rotatively disposed a drive bushing 68 having an inner bore within which crank pin 32 is drivingly disposed. Crank pin 32 has a flat on one surface (not shown) which drivingly engages a flat surface in a portion of the inner bore of drive bushing 68 to provide a radially compliant drive arrangement such as shown in assignee's U.S. Pat. No. 4,877,382, the disclosure of which is incorporated herein by reference.

Please replace the paragraph beginning at Col. 3, Line 55, with the following paragraph:

[0001] Non-orbiting scroll member 74 has a centrally disposed discharge port 88 which is in fluid communication via an opening 90 in partition 20 with a discharge muffler 92 defined by cap 14 and partition 20. Fluid compressed by the moving pockets between scroll wraps 60 and 72 discharges into discharge muffler 92 through port 88 and opening 90. Non-orbiting scroll member 74 has in the upper surface thereof an annular recess 94 having parallel coaxial sidewalls within which is sealingly disposed for relative axial movement an annular seal assembly 96 which serves to isolate the bottom of recess 94 so that it can be placed in fluid communication with a source of intermediate fluid pressure by means of a passageway 98. Non-orbiting scroll member 74 is thus axially biased against orbiting scroll member 56 by the forces created by discharge pressure acting on the central portion of non-orbiting scroll member 74 and the forces created by intermediate fluid pressure acting on the bottom of recess 94. This axial pressure biasing, as well as the various techniques for supporting non-orbiting scroll member 74 for limited axial movement, are disclosed in much greater detail in assignee's aforementioned U.S. Pat. No. 4,877,382.

Please replace the paragraph beginning at Col. 4, Line 10, with the following paragraph:

Relative rotation of scroll members 56 and 74 is prevented by the usual Oldham Coupling 100 having a pair of keys slidably disposed in diametrically opposing slots in non-orbiting scroll member 74 and a second pair of keys slidably disposed in diametrically opposed slots in orbiting scroll member 56.

Please replace the paragraph beginning at Col. 5, Line 51, with the following paragraph:

Valve guide support 234 is attached to an adjacent leg of main bearing housing 24 and it defines a bore 248 which slidingly receives slider valve 232 and guides its movement. Valve return spring 236 is located between valve guide support 234 and slider valve 232 to bias slider valve 232 into its vapor injection position as shown in FIG.

4. Activating fitting 238 is in communication with one end of bore 240 through a bore 250 in fitting 238, a port 252 in shell 12 and a passage 254 in the leg of main bearing housing 24. Bore 250 is connected to a source of pressurized fluid, such as the discharge pressure of the compressor, through a valve such as a solenoid valve. When this pressurized fluid is provided to the end of bore 240, slider valve 232 moves from its position shown in FIG. 4 to a position where modulation slot 246 aligns with fluid passage 118 to permit modulation of the capacity of the compressor through a port 260 extending through main bearing housing 24. A seal 256 isolates the pressurized fluid provided through activating fitting 238. When the vapor injection feature is again desired, the pressurized fluid can be released from fitting 238 allowing valve return spring 236 to again align vapor injection through hole with passage 118 as shown in FIG. 4.

Please replace the paragraph beginning at Col. 6, Line 15, with the following paragraph:

Fluid passages 314 each extend generally vertical from thrust bearing surface 54 to the internal suction area of shell 12. Each fluid passage 314 comprises counter bored portion 124 which opens up on thrust bearing surface 54. Counter bore portions 124 maintain communication with their respective injection hole 112 during all movement of orbiting scroll member 56. The lower ends of fluid passages 314 each define an enlarged bore 324 which mates with a respective tubing assembly 316.

Please replace the paragraph beginning at Col. 6, Line 32, with the following paragraph:

Tubing connector assembly 318 comprises a main bearing housing fitting 340 and a connecting tube 342. Fitting 340 is secured to main bearing housing 24 by a plurality of bolts. Fitting 340 defines an internal bore 344 which is in communication with the pair of tubes 328. Connecting tube 342 is disposed within bore 344 of fitting and extends to fluid injection fitting 322. A seal 346 seals the interface between tube 342 and bore 344.

Applicants submit herewith a separate sheet containing the Abstract of the disclosure pursuant to 37 CFR § 1.52(b)(4). The Abstract is reproduced below as well as in the attached separate sheet.

ABSTRACT

A scroll compressor includes a fluid injection system. The fluid injection system includes a fluid passage which extends from a position outside of the shell of the compressor to the fluid pockets created by the wraps of the scroll members. The fluid passage extends through the orbiting scroll member. The fluid injection system can be used to inject gaseous or liquid refrigerant, lubricant or the fluid injection system can be used to vent the moving pocket to the suction area of the compressor for capacity modulation.